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Three-Dimensional Boundary layer Flow and Heat Transfer of a Fluid Particle Suspension over a Stretching Sheet Embedded in a Porous Medium

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Abstract: This article presents the effect of nonlinear thermal radiation on three dimensional flow and heat transfer of fluid particle suspension over a stretching sheet. The combined effects of non-uniform source/sink and convective boundary condition are taken into consideration. The governing partial differential equations are transformed into ordinary differential equations using similarity variables, which are then solved numerically by using Runge Kutta Fehlberg-45 method with shooting technique. The influence of various parameters on velocity and temperature profiles are illustrated graphically, and discussed in detail. The results indicate that the fluid phase velocity is greater than that of the particle phase for various existing parameters.

Keywords: Convective boundary condition; Fluid particle suspension; nonlinear thermal radiation; Stretching sheet; Three dimensional flow

1 Introduction

Heat transfer phenomenon due to suspended particles into the fluid has important role in recent and advanced processes of industrial and engineering problems concerned with powder technology, sedimentation, rain erosion in guided missiles, combustion, atmospheric fallout, fluidization, nuclear reactor cooling, electrostatic precipi-

tation of dust, waste water treatment, acoustics batch settling and so forth. A study on fundamentals of dusty fluid was made by Saffman [1]. Vajravelu and Nayfeh [2] have investigated the hydromagnetic flow of a dusty fluid over a porous stretching sheet. Some more investigations on heat transfer process with fluid particle suspension can be seen in the Refs [3–9]. Al-Rashed et al. [10] report the influence of surface waviness on natural convection boundary layer flow of the two-phase dusty fluid having compressible nature. Recently the effect of thermal stratification on MHD flow and heat transfer of dusty fluid over a vertical stretching sheet embedded in a thermally stratified porous medium in the presence of uniform heat source and thermal radiation has been numerically investigated by Gireesha et al. [11].

Thermal radiation plays an important role in manufacturing industries for the design of nuclear power plants and several engineering applications. Due to its vital applications, numerous researchers have paid their attention to thermal radiation effect [12–17]. Mahanthesh et al. [18] investigated the Marangoni transport of dissipating SWCNT and MWCNT nanofluids under the influence of magnetic force and radiation. Further, it is worth to notice that the linear radiation is valid for small temperature difference. But, for the larger temperature difference nonlinearized Rosseland approximation is to be considered. Hayat et al. [19] initiated the tangent hyperbolic nanofluid flow in the presence of nonlinear thermal radiation. The idea of nonlinear thermal radiation along with heat transfer phenomenon has recently been presented by so many researchers (see [20–23]). Recently, Prasannakumara et al. [24] studied the effect of nonlinear thermal radiation on slip flow and heat transfer of fluid particle suspension with nanoparticles over a nonlinear stretching sheet immersed in a porous medium.

Three dimensional flow has many applications in solar collectors, aeronautical engineering, science and technology, crude oil purification, magnetic material processing, geophysics and controlling of cooling rate, insulation engineering, grain storage devices, ground water pollution, purification process and petroleum reservoirs.

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$$u_p \frac{\partial w_p}{\partial x} + v_p \frac{\partial w_p}{\partial y} + w_p \frac{\partial w_p}{\partial z} = \frac{1}{\tau_v} (w - w_p), \quad (7)$$

$$u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} + w \frac{\partial T}{\partial z} = \frac{\partial}{\partial z} \left[\left(k + \frac{16\sigma^* T_\infty^3}{3k^*} \right) \frac{\partial T}{\partial z} \right] \frac{1}{\rho c_p} + \frac{\rho_p}{\rho} \frac{T_p - T}{\tau_T} + \frac{\rho_p}{\rho c_p} \frac{1}{\tau_v} \left[(u_p - u)^2 + (v_p - v)^2 \right] + q''' \quad (8)$$

$$u_p \frac{\partial T_p}{\partial x} + v_p \frac{\partial T_p}{\partial y} + w_p \frac{\partial T_p}{\partial z} = \frac{c_p}{c_m} \frac{T_p - T}{\tau_T}, \quad (9)$$

with boundary conditions as [8];

$$\begin{aligned} u = u_w, \quad v = v_w, \quad w = 0, \quad -k \frac{\partial T}{\partial y} &= h_f(T_f - T) \text{ at } z = 0, \\ u_p = u = 0, \quad v_p = v = 0, \quad w_p = w, \quad \rho_p &= \rho\omega, \\ T = T_\infty, \quad T_p = T_\infty \text{ as } z \rightarrow \infty, \end{aligned} \quad (10)$$

where (u, v, w) and (u_p, v_p, w_p) denote the respective velocity components of the fluid and dust phases along the x, y and z -directions. ρ and ρ_p are the density of the fluid and dust phase respectively. k, ν, c_p and c_m are thermal conductivity, kinematic viscosity, the specific heat of fluid and dust phase, respectively. τ_T is the thermal equilibrium time i.e., the time required by the dust cloud to adjust its temperature to the fluid, τ_v is the relaxation time of the of dust particle i.e., the time required by a dust particle to adjust its velocity relative to the fluid. T and T_p represents the temperatures of the fluid and dust particles inside the boundary layer respectively. Throughout the study, it is assumed that, $c_p = c_m$. In deriving these equations, the drag force is considered for the interaction between the fluid and particle phases. h_f is the convective heat transfer coefficient, T_f is the convective fluid temperature below the moving sheet. Further, q''' is the space and temperature dependent internal heat generation/absorption (non-uniform heat source/sink) which can be expressed as;

$$q''' = \left(\frac{ku_w(x)}{xv} \right) \left[A^* (T_w - T_\infty) f'(\eta) + B^* (T - T_\infty) \right], \quad (11)$$

where A^* and B^* are the parameters of the space and temperature dependent internal heat generation/absorption. It is to be noted that, A^* and B^* are positive for internal heat source and negative for internal heat sink.

Introduce the following similarity transformations to reduce the partial differential equations in to set of ordinary ones;

$$\begin{aligned} u &= c\lambda x f'(\eta), \quad v = cy \left[f'(\eta) + g'(\eta) \right], \\ w &= -\sqrt{cv} \left[g(\eta) + (\lambda + 1)f(\eta) \right], \quad \eta = \sqrt{\frac{c}{\nu}} z, \end{aligned}$$

$$\begin{aligned} u_p &= c\lambda x F(\eta), \quad v_p = cy \left[F(\eta) + G(\eta) \right], \\ w_p &= \sqrt{cv} \left[G(\eta) + (\lambda + 1)K(\eta) \right], \end{aligned}$$

$$\rho_r = \frac{\rho_p}{\rho} = H(\eta). \quad (12)$$

Here prime denotes differentiation with respect to η . Making use of (12) in equations (1) to (7), continuity equations (1) and (4) are identically satisfied and the remaining momentum equations take the following form:

$$f''' + [g + (\lambda + 1)f]f'' - f'^2\lambda + \beta(F - f')H - k_p f' = 0, \quad (13)$$

$$(f''' + g''') + [g + (\lambda + 1)f](f'' + g'') - (f' + g')^2 +$$

$$\beta[(F + G) - (f' + g')] - k_p[f' + g'] = 0, \quad (14)$$

$$[G + (\lambda + 1)K]F' + \lambda F^2 + \beta(F - f') = 0, \quad (15)$$

$$\begin{aligned} [F + G]^2 + [G + (\lambda + 1)K][F' + G'] \\ + \beta[(F + G) - (f' + g')] = 0, \end{aligned} \quad (16)$$

$$\begin{aligned} [G + (\lambda + 1)K][(\lambda + 1)K' + G'] \\ + \beta[G + g + (\lambda + 1)(K + f)] = 0, \end{aligned} \quad (17)$$

$$[G + (\lambda + 1)K]H' + [(\lambda + 1)(F + K') + G + G']H = 0. \quad (18)$$

Corresponding boundary conditions becomes;

$$\text{At } \eta = 0 : f' = 1, \quad f = 0, \quad g = g' = 0.$$

$$\text{A } \eta \rightarrow \infty : f' = F = 0, \quad g' = G = 0, \quad K = -f - \frac{g}{\lambda + 1}, \quad H = \omega. \quad (19)$$

where, $\beta = 1/c\tau_v$ is the fluid-particle interaction parameter, $H = \rho_p/\rho$ is the relative density and $k_p = \frac{\nu}{k^*c}$ is the permeability parameter and ω is the density ratio and is considered as 0.2 in this present study.

2.1 Heat transfer solution

To transform the energy equations into a non-dimensional form, dimensionless temperature profile for the clean fluid and dusty fluid are introduced as follows:

$$\theta(\eta) = \frac{T - T_\infty}{T_f - T_\infty}, \quad \theta_p(\eta) = \frac{T_p - T_\infty}{T_f - T_\infty} \quad (20)$$

where, T_∞ denotes the temperature at larger distance from the wall with $T = T_\infty(1 + (\theta_w - 1)\theta)$ and $\theta_w = \frac{T_f}{T_\infty}$ being the temperature ratio parameter. Making use of equations (12) and (20) in equations (8) and (9) the energy equations takes the following form:

$$\left((1 + Rd(1 + (\theta_w - 1)\theta)^3) \theta' \right)' + \text{Pr}[g + (\lambda + 1)f] \theta'$$

$$+ Pr\beta_\tau [\theta_p - \theta] H + Pr\beta \left[Ec_x (F - f')^2 + Ec_y (F - f' + G - g')^2 \right] H + \lambda (A^* f' + B^* \theta) = 0, \quad (21)$$

$$[G + (\lambda + 1) + K] \theta_p' + \frac{C_p}{c_m} \beta_\tau [\theta_p - \theta] = 0. \quad (22)$$

where, $Rd = \frac{16\sigma^* T_\infty^3}{3kk^*}$ is the radiation parameter, $Pr = \frac{\mu c_p}{k}$ is the Prandtl number, $Ec_x = \frac{(u_w)^2}{(T_f - T_\infty)c_p}$ and $Ec_y = \frac{(v_w)^2}{(T_f - T_\infty)c_p}$ are the Eckert numbers and $\beta_\tau = \frac{1}{\tau_{\tau c}}$ is the fluid-particle interaction parameter for temperature. The corresponding boundary conditions take the following form;

$$\eta = 0 : \theta' = -Bi(1 - \theta), \quad (23)$$

$$\eta \rightarrow \infty : \theta_p = \theta = 0.$$

where $Bi = \sqrt{\frac{v}{c}} \frac{h_f}{k}$ is the Biot number.

The wall shear stress is given by,

$$\tau_{zx} = -\mu \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)_{z=0}, \quad \tau_{zy} = -\mu \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right)_{z=0}.$$

The friction factor is written as,

$$Cf_x Re_x^{\frac{1}{2}} = -f''(0), \quad Cf_y Re_y^{\frac{1}{2}} = -g''(0).$$

The surface heat transfer rate is given by,

$$q_w = -k \left(\frac{\partial T}{\partial y} \right)_{z=0} + (q_r)_w.$$

The local Nusselt number is written as

$$Nu Re_x^{-\frac{1}{2}} = - \left(1 + Rd \theta_w^3 \right) \theta'(0).$$

3 Numerical solution

The set of non-linear differential equations (13–18) and (21–22) with boundary conditions (19) and (23) have been solved using Runge-Kutta-Fehlberg fourth-fifth order method along with shooting technique. In the first step, equations of higher order are discretized to a system of simultaneous differential equations of first order by introducing new dependent variables. Missed initial conditions are obtained with the help of shooting technique. Afterward, a finite value for η_∞ is chosen in such a way that all the far field boundary conditions are satisfied asymptotically. Our bulk computations are considered with the value at $\eta_\infty = 5$, which is sufficient to achieve the far field boundary conditions asymptotically for all values of the parameters considered.

4 Results and discussion

Numerical computation has been carried out in order to study the influence of different parameters that describes the flow and heat transfer characteristics of dusty fluid. The results are presented and discussed in detail through graphical representation and tables.

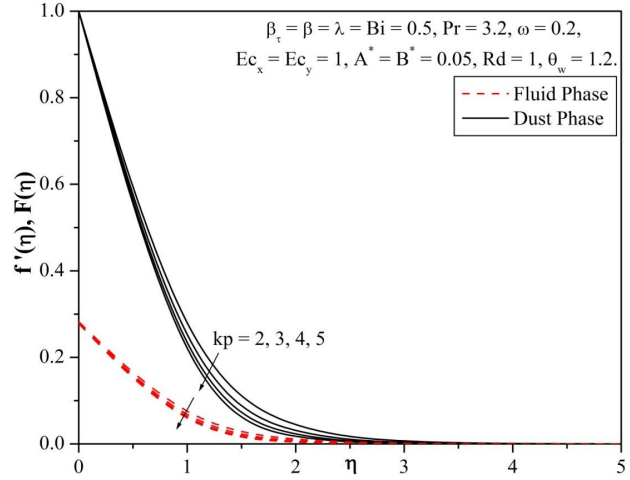


Fig. 2: Dimensionless profiles of u , u_p velocity components for different values of k_p .

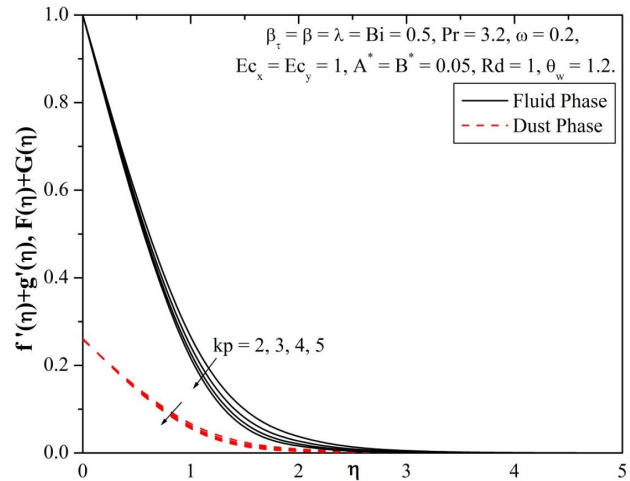


Fig. 3: Dimensionless profiles of v , v_p velocity components for different values of k_p .

The dimensionless velocity profiles for different values of k_p proportional to u and v velocity components are depicted in Figure 2 and 3, respectively for both fluid and dust phase. Here $f'(\eta)$ represents the velocity in x -direction while $[f'(\eta) + g'(\eta)]$ is the velocity along

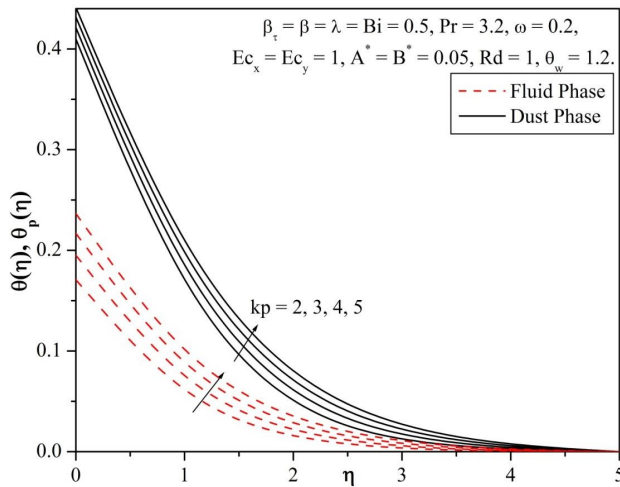


Fig. 4: Dimensionless temperature profiles for different values of k_p .

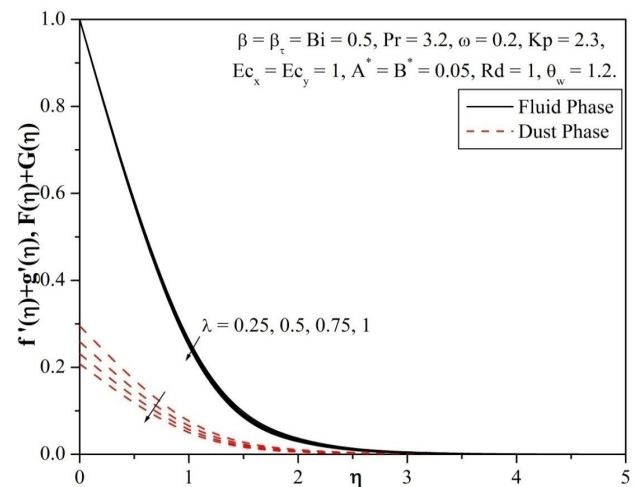


Fig. 6: Dimensionless profiles of v , v_p velocity components for different values of λ .

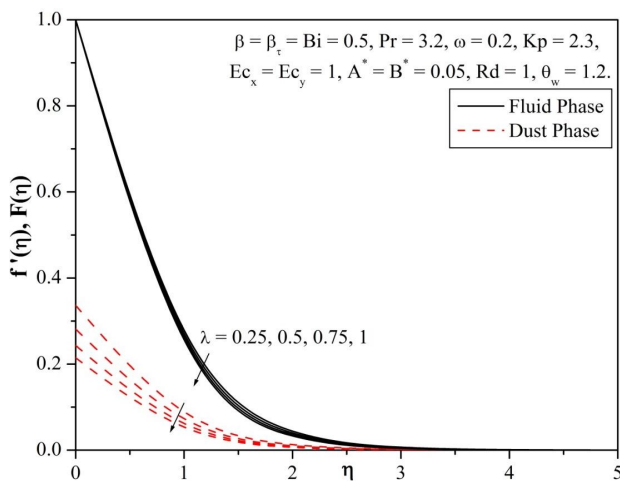


Fig. 5: Dimensionless profiles of u , u_p velocity components for different values of λ .

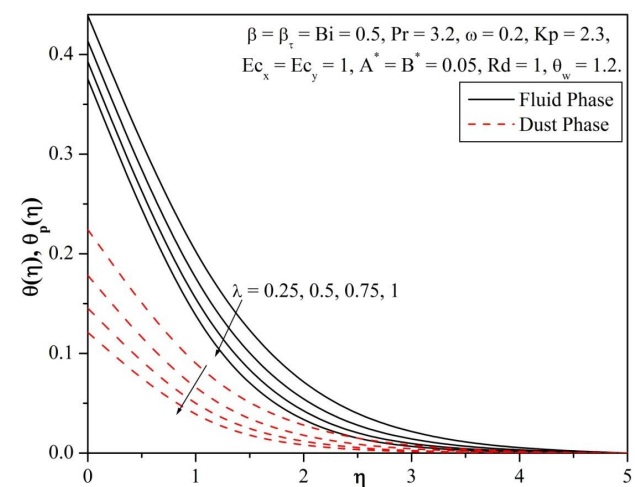


Fig. 7: Dimensionless temperature profiles for different values of λ .

y -direction for fluid phase. Further, $F(\eta)$ represents the velocity in x -direction while $[F(\eta) + G(\eta)]$ is the velocity in y -direction for dust phase respectively. From these figures, it is observed that both the velocities are retarded for the case of increasing k_p . This is because, an increase in permeability parameter causes the resistance to the fluid motion and hence velocity decreases for both fluid and dust phase. The effect of increasing values of permeable parameter contributes to the thickening of thermal boundary layer, and is shown in Figure 4. This is evident from the fact that, the porous medium opposes the fluid motion. The resistance offered to the flow is responsible in enhancing the temperature.

The effect of various values of λ for dimensionless velocity profiles are depicted in Figure 5 and 6 for both fluid and dust phase. These figures portraits that, the behavior

of the $f'(\eta)$, $F(\eta)$, $f'(\eta) + g'(\eta)$ and $F(\eta) + G(\eta)$ are the same and it decreases with increase in values of λ . It can also be seen that, fluid phase velocity is greater than dust phase velocity. The variation of dimensionless temperature profile for both phases along with different values of λ , are illustrated in Figure 7. From this figure it is seen that, increase of λ causes decrease of the temperature profile of both dusty and fluid phase. Furthermore, one can observe from this figure that values of the temperature are higher for clean fluid than for the dusty fluid at all points, as expected.

Figures 8 and 9 are plotted to view the effect of fluid particle interaction parameter β on velocity profile respectively in both directions for fluid and dust phase. From these figures it is examined that, the velocity u , v decays while u_p , v_p is enhanced for larger values of β .

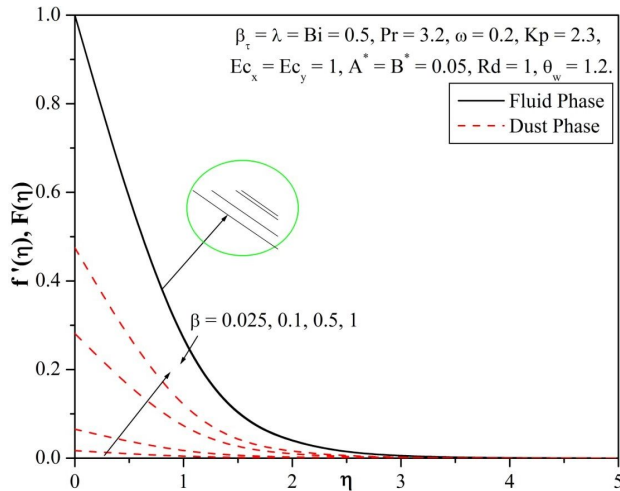


Fig. 8: Dimensionless profiles of u , u_p velocity components for different values of β .

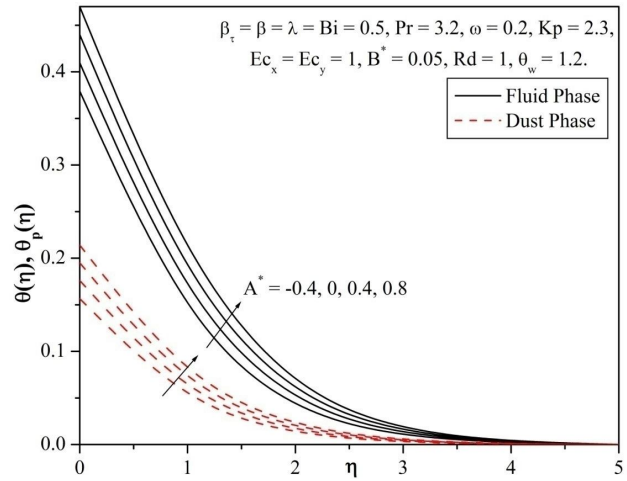


Fig. 10: Dimensionless temperature profiles for different values of A^* .

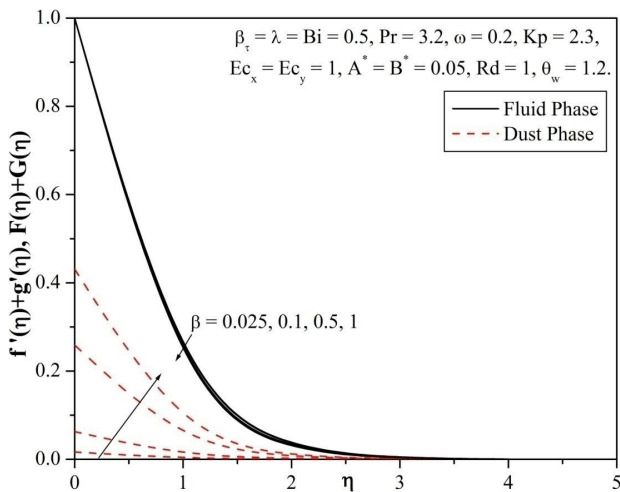


Fig. 9: Dimensionless profiles of v , v_p velocity components for different values of β .

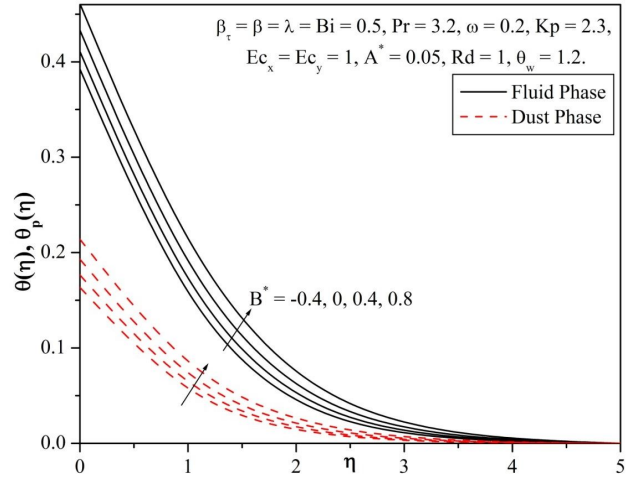


Fig. 11: Dimensionless temperature profiles for different values of B^* .

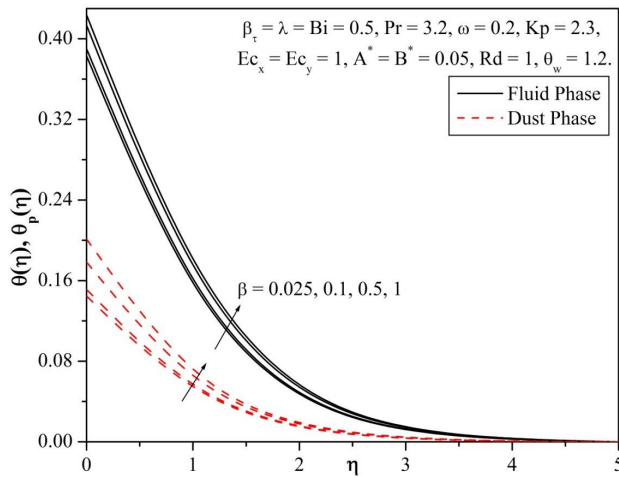
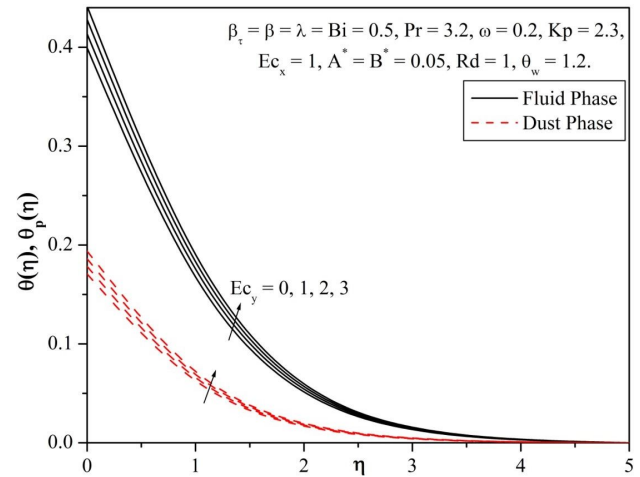
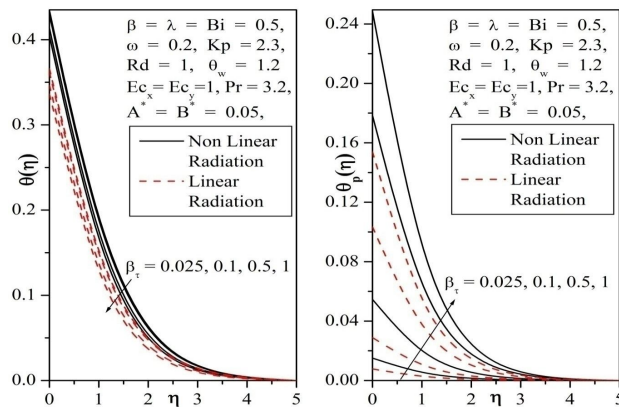
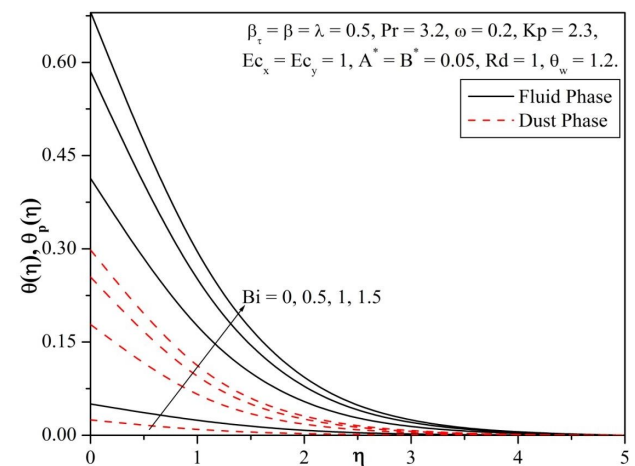
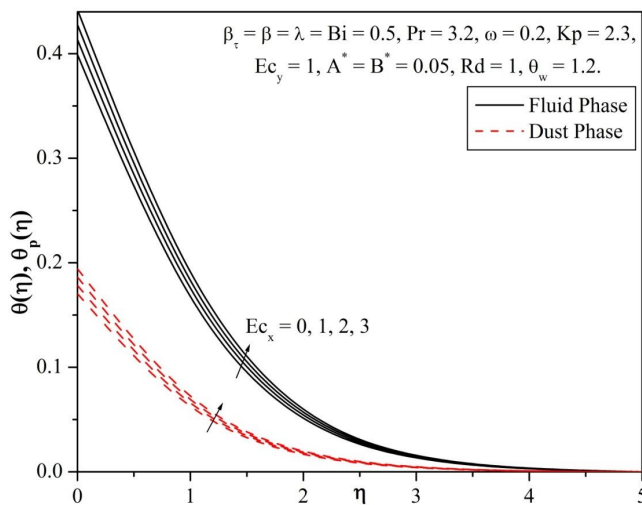
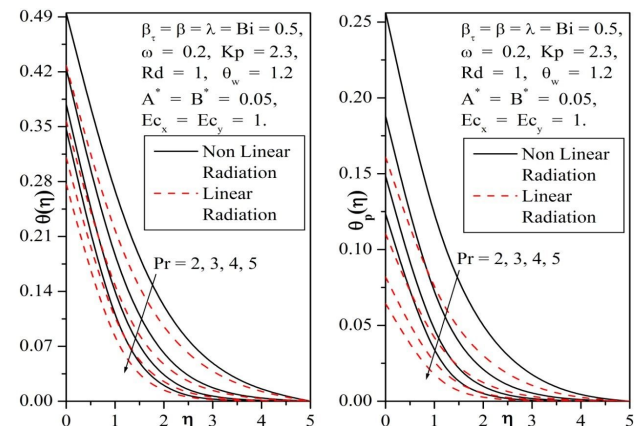
The variation of dimensionless temperature profiles for different values of space-dependent heat source/sink parameter A^* and temperature-dependent heat source/sink parameter B^* are plotted in Figures 10 and 11 respectively. It is evident from these graphs that, increasing A^* and B^* results in the enhancement of both fluid phase and dust phase temperature.

Figure 12 and 13 characterize the temperature profiles for distinct values of fluid and thermal particle interaction parameter β and β_τ respectively. Figure 12 indicates that the temperature of both phases increases with increases in β and it reveals that, effect of variation of β is more sensible on dusty phase than for the fluid phase. This is because of the direct effect of β on velocity and since the temperature depends on velocity then the temperature varies with

variation of β . Effect of β_τ on fluid and dust phase temperature profiles for both linear and nonlinear thermal radiation cases are shown in Figure 13. It can be seen that fluid phase temperature decreases and dust phase temperature increases with increase in β_τ for both the cases. It can also be seen that nonlinear radiation is more influential than linear radiation.

Figures 14 and 15 illustrate the effect of the Eckert number (Ec) on temperature distribution. These figures show that the increasing values of Eckert number is to increase the temperature distribution. This is due to the fact that, heat energy is stored in the liquid due to frictional heating which results into increasing in its temperature and this is true in both the cases.

Figure 16 resemble the change in temperature profile for different values of Biot number (Bi). It describes that

Fig. 12: Dimensionless temperature profiles for different values of β .Fig. 15: Dimensionless temperature profiles for different values of Ec_y .Fig. 13: Dimensionless temperature profiles for different values of β_τ .Fig. 16: Dimensionless temperature profiles for different values of Bi .Fig. 14: Dimensionless temperature profiles for different values of Ec_x .Fig. 17: Dimensionless temperature profiles for different values of Pr .

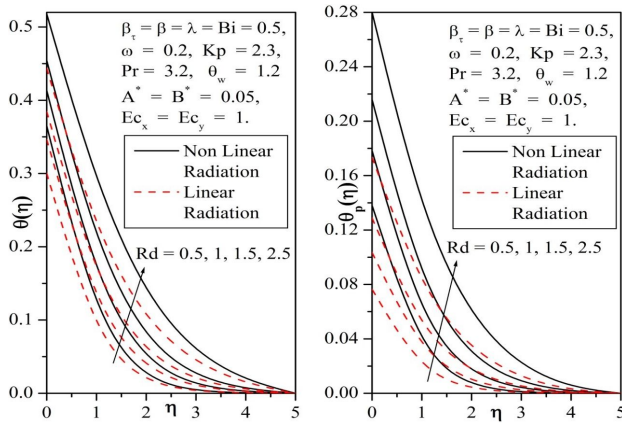


Fig. 18: Dimensionless temperature profiles for different values of Rd .

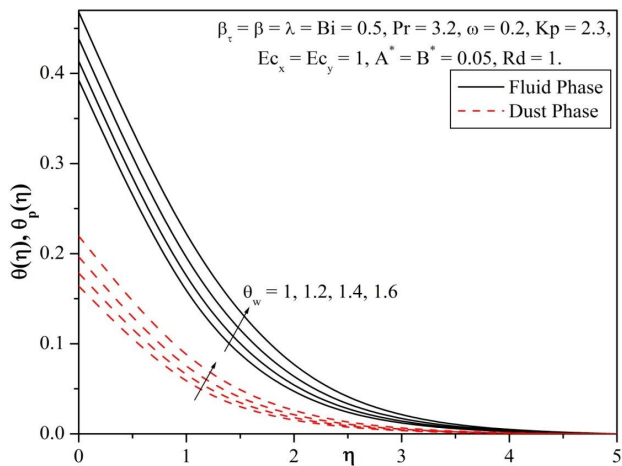


Fig. 19: Dimensionless temperature profiles for different values of θ_w .

the increasing values of Biot number lead to elevated temperature and thicker thermal boundary layer. This is due to the fact that, the convective heat exchange at the surface leads to enhance the thermal boundary layer thickness.

Figure 17 is sketched for the temperature distribution against the Prandtl number (Pr). From this figure, it reveals that the temperature decreases with increase in the value of Pr . This is because, as the Prandtl number increases, thermal diffusivity decreases there by decreases the temperature. Hence Prandtl number can be used to increase the rate of cooling. Figure 18 exhibits the variation of temperature profile for the radiation parameter. As, the radiation parameter releases the heat energy into the flow, with an increase of radiation parameter, temperature profile increases.

The effect of temperature ratio parameter (θ_w) over the dimensionless temperature is shown in Figure 19. It is observed that, the temperature profile increases for both

phases for the increasing the values of θ_w . This is because, fluid temperature is much higher than the ambient temperature for increasing values of θ_w , which increases the thermal state of the fluid. From the obtained result it can be concluded that, non linear radiation has more influence as compared to linear radiation.

Table 1: Values of skin friction coefficient and Nusselt number for K_p and β .

K_p	β	$Cf_x Re_x^{-\frac{1}{2}}$	$Cf_y Re_y^{-\frac{1}{2}}$	$Nu Re_x^{-\frac{1}{2}}$
2		1.80414	2.48579	0.868129
3		2.06244	2.66918	0.851712
4		2.29192	2.84405	0.836969
	0.1	1.87066	2.40828	0.850894
	0.4	1.88209	2.51217	0.860442
	0.7	1.89111	2.59441	0.867504

Table 2: Values of Nusselt number for various existing parameters

A^*	B^*	Bi	Ec_x	Ec_y	Pr	Rd	θ_w	β_τ	$Nu Re_x^{-\frac{1}{2}}$
0	0.05	0.5	1	1	3.2	1	1.2	0.5	0.864929
0.4									0.849573
0.8									0.834262
	0								0.863836
	0.4								0.857119
	0.8								0.32066
		0.5							0.863007
		1							1.232436
		1.5							1.426802
			0						0.863576
			0.5						0.863292
			1.5						0.862723
				0					0.865188
				0.5					0.864098
				1.5					0.861915
					3				0.846785
					4				0.916951
					5				0.966899
						0.5			0.641931
						1.5			1.052475
						2.5			1.368408
							1		0.653644
							1.4		1.137426
							1.6		1.471094
								0.1	0.839874
								0.4	0.858548
								0.7	0.870296

Variations of skin friction coefficient and local Nusselt number for the governing parameters are recorded in the Table 1 and 2. From these table, it is clear that skin friction coefficient in x and y direction increases with K_p and β . Further, Nusselt number decreases for K_p and increases

for β . Moreover, Nusselt number are increasing functions of Pr , β_T , B_i while it decreases for A^* , B^* , Rd , θ_w and Ec .

5 Conclusion

In the present study, three-dimensional boundary layer flow and heat transfer of a dusty fluid towards a stretching sheet embedded in a porous medium in presence of non-uniform source/sink with convective boundary condition is investigated. By using the appropriate transformation for the velocity and temperature, the basic equations governing the flow and heat transfer were reduced to a set of ordinary differential equations. These equations are solved numerically using the fourth-fifth-order Runge-Kutta-Fehlberg method. Some of the conclusion obtained from this investigation are summarized as follows:

1. Fluid phase velocity is always greater than that of the particle phase.
2. Velocity of fluid and dust phases decrease with increases in permeability parameter.
3. Velocity and temperature profile of fluid phase and dust phases decrease with increases in velocity ratio parameter λ .
4. Increase of β will decrease fluid phase velocity and increases dust phase velocity.
5. Increase of β_T will decrease fluid phase and increases the dust phase of temperature profile.
6. Fluid phase temperature is higher than the dust phase temperature.
7. Temperature profiles of fluid and dust phases increases with the increase of the Ec , A^* , B^* , Bi , Rd and θ_w .

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