

# LIQUID CIRCULATION, GAS HOLDUP AND PRESSURE DROP IN BUBBLE COLUMN WITH DRAUGHT TUBE

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The effects of gas velocity and geometrical dimensions of apparatus on the flow rate  $Q_L$  of circulating liquid, gas holdups and pressure drops in bubble columns with draught tube were examined. It was found that  $Q_L$  increases with increasing gas velocity, column diameter, draught tube length, and height of the lower end of draught tube, and decreases with increasing liquid viscosity. It was also found that the maximum value of  $Q_L$  is observed for constant values of gas velocity and column diameter, when the diameter ratio of draught tube and column is about 0.6.

Empirical equations for gas holdups in draught tube and annulus and the pressure drops due to flow reversals, necessary for estimating  $Q_L$ , are proposed.

## Introduction

The bubble column with draught tube is widely used in various processes,<sup>2)</sup> including chemical, fermentation, leaching and waste water treatment processes. Gas is bubbled into a draught tube, and the liquid flows upwards in the tube and downwards in the annulus by air-lift action.

Many studies<sup>4,6-8,11)</sup> of this circulating liquid flow have been made, but the results of flow rate estimation of circulating liquid proposed in these studies do not agree with each other. This disagreement might be due to the difference of experimental techniques used for measuring liquid flow rate or inadequate assumptions used in deriving the equation for estimating liquid flow rates.

The purpose of this work is to observe experimentally the effects of gas velocity  $U_G$  and geometrical dimensions of apparatus on the flow rate of circulating liquid and to establish a method of estimating the flow rate in a bubble column with draught tube into which gas is dispersed.

## 1. Fundamentals

Figure 1 shows a scheme of the bubble column with draught tube.

Macroscopic mass, momentum and energy bal-

ances in the column give inherent relationships among the flow rate of circulating liquid, liquid velocities, gas holdups and reversal losses due to reversing flow direction. A mass balance between section 1-4 in Fig. 1 gives

$$Q_L = S_a \bar{U}_{La} = S_i \bar{U}_{Li} = S_o \bar{U}_{Lc}/2 \quad (1)$$

where

$$S_a = \pi \{ D_o^2 - (D_i + 2t_w)^2 \} / 4, \quad S_i = \pi D_i^2 / 4$$

and

$$S_o = \pi D_o^2 / 4.$$

Momentum and energy balances among sections 1-4 in Fig. 1 give the following equations for  $H_N \leq L$ .

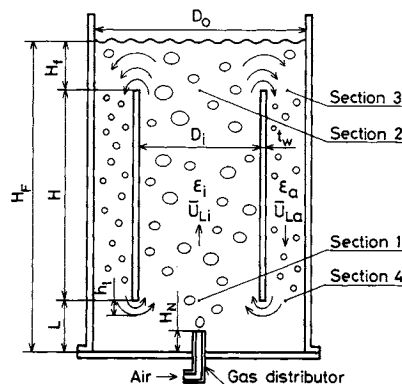


Fig. 1. Scheme of bubble column with draught tube and with gas dispersion into it.

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$$P_1 - P_2 = (4H\tau_{wi}/D_i) + \rho_L g H(1 - \varepsilon_i) \quad (2)$$

$$P_2 - P_3 = \Delta P_u + \frac{\rho_L}{2} \left[ \left( \frac{\bar{U}_{La}}{1 - \varepsilon_a} \right)^2 - \left( \frac{\bar{U}_{Li}}{1 - \varepsilon_i} \right)^2 \right] \quad (3)$$

$$P_3 - P_4 = (4H\tau_{wa}/D_e) - \rho_L g H(1 - \varepsilon_a) \quad (4)$$

$$P_4 - P_1 = \Delta P_t + \frac{\rho_L}{2} \left[ \left( \frac{\bar{U}_{Li}}{1 - \varepsilon_i} \right)^2 - \left( \frac{\bar{U}_{La}}{1 - \varepsilon_a} \right)^2 \right] \quad (5)$$

Adding Eqs. (2)–(5) gives

$$\rho_L g H(\varepsilon_i - \varepsilon_a) = \Delta P_t + \Delta P_u + \Delta P_{fa} + \Delta P_{fi} \quad (6)$$

where

$$\Delta P_{fa} = 4H\tau_{wa}/D_e \quad (7)$$

$$\Delta P_{fi} = 4H\tau_{wi}/D_i \quad (8)$$

For  $H_N > L$  in Fig. 1, the equation corresponding to Eq. (6) is given by Eq. (9).

$$\begin{aligned} & \rho_L g \varepsilon_i (H + L - H_N) - \rho_L g H \varepsilon_a \\ & = \Delta P_t + \Delta P_u + \Delta P_{fa} + \Delta P_{fi} \end{aligned} \quad (9)$$

where

$$\Delta P_{fi} = \frac{4\tau_{w0}(H_N - L)}{D_i} + \frac{4\tau_{wi}(H + L - H_N)}{D_i} \quad (10)$$

Single-phase liquid flow is assumed in the region of  $L \leq y \leq H_N$  as indicated in the first term of the right side of Eq. (10), though bubbles are entrained from the annulus by liquid flow and exist in this region.

$\varepsilon_a$ ,  $\varepsilon_i$ ,  $\Delta P_t$ ,  $\Delta P_u$ ,  $\Delta P_{fa}$  and  $\Delta P_{fi}$  in Eqs. (6) and (9) must be given as functions of average gas velocity  $\bar{U}_G$ ,  $Q_L$ , the properties of the liquid and the geometrical dimensions of the column so that  $Q_L$ , which is related to  $\bar{U}_{Li}$  and  $\bar{U}_{La}$  through Eq. (1), can be estimated by Eq. (6) or Eq. (9).

## 2. Experimental

### 2.1 Experimental apparatus and conditions

The experimental apparatus used in this work is shown in Fig. 2. Three kinds of plexiglass columns with flat bottom were used. Their dimensions are 0.1 m dia., 1 m in height, 0.14 m dia., 1 m and 2 m in height, and 0.30 m dia., 2 m in height. Table 1 shows the dimensions of draught tubes and their height from the base plate of the column. Single orifice plates, single nozzles and perforated plates were used as gas distributors, details of which are shown in Table 2. The gas velocity  $\bar{U}_G$  of air used was  $0.00192$ – $0.128 \text{ m} \cdot \text{s}^{-1}$ .

The liquids used were  $10 \text{ mol} \cdot \text{m}^{-3}$  KCl aqueous solution and  $100 \text{ mol} \cdot \text{m}^{-3}$  KCl–50 wt% glycerol aqueous solution at a temperature of 293.2 K. The height of liquid without gas flowing was kept at 0.741–1.55 m above the base plate of the column. During a run, the liquid was neither fed nor dis-

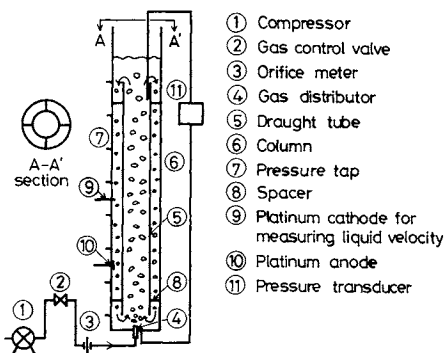


Fig. 2. Experimental apparatus.

Table 1. Experimental apparatus

$D_o$ [m]	$D_i$ [m]	$t_w$ [mm]	$H$ [m]	$L/D_i$ [—]	$H_L$ [m]	Gas distributors*
0.100	0.052	3	0.70	0.50	0.84	S1
0.100	0.066	3	0.70	0.50	0.84	S1
0.140	0.066	3	0.70	0.50	0.84	S2, S3, P1, P2
0.140	0.078	5	0.70	0.50	0.84	S2
0.140	0.082	3	0.70	0.24–1.0	0.74–0.94	S2, S3, P1
0.140	0.082	3	0.70	0.24–1.0	0.74–0.94	P2, N1, N2
0.140	0.082	3	1.40	0.50	1.54	S2, S3
0.140	0.082	2	0.70	0.50	0.84	S2, S3
0.140	0.094	3	0.70	0.50	0.84	S3
0.140	0.104	3	0.70	0.50	0.84	S3
0.300	0.190	5	1.40	0.50	1.55	N3

\* Details are shown in Table 2.

Table 2. Details of gas distributors

Key	Gas distributor	$\delta$ [mm]	$N$ [—]	$H_N$ [m]
S1	Single orifice plate	2.2	1	0
S2	Single orifice plate	3.0	1	0
S3	Single orifice plate	3.6	1	0
N1	Single nozzle	4.0	1	0.041
N2	Single nozzle	4.0	1	0.216
N3	Single nozzle	15.0	1	0
P1	Perforated plate	0.5	19	0
P2	Perforated plate	1	7	0

charged. Table 3 shows the properties of the liquid at operating temperature.

### 2.2 Measurement of liquid flow rate

The flow rate of circulating liquid was obtained by measuring the liquid velocities  $U_{La}$  in the annulus.  $U_{La}$  was obtained by measuring diffusion current  $i_d$  of dissolved oxygen on a fine spherical platinum cathode. In this work potassium chloride was used as a supporting electrolyte, and in this case the following reaction occurs on the platinum cathode:

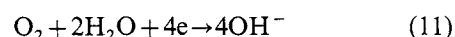


Figure 3 shows details of the platinum cathode, which is insulated electrically excepts for its spherical

Table 3. Properties of liquids

Liquid	$\rho_L$ [kg·m <sup>-3</sup> ]	$\mu_L \times 10^3$ [Pa·s]	$\sigma_L \times 10^3$ [N·m <sup>-1</sup> ]
10 mol·m <sup>-3</sup> KCl aq. soln.	998.2	1.01	72.6
100 mol·m <sup>-3</sup> KCl – 50 wt% Glycerol aq. soln.	1134	6.09	68.0

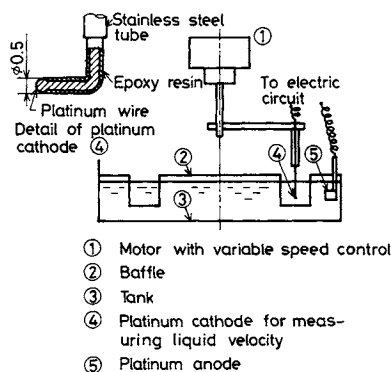


Fig. 3. Experimental apparatus of calibrating platinum cathode for measuring liquid velocity.

tip and is supported with a stainless steel tube 2 mm in diameter. An experimental equipment for calibrating the relation of  $i_d$  vs. liquid velocity on this spherical electrode is also shown in Fig. 3. After the liquid in the tank (3 in Fig. 3) was saturated with air, the spherical electrode installed on an arm fitted to a motor was rotated at a constant speed in the liquid, and a certain voltage where a diffusion current  $i_d$  was observed was applied between the spherical cathode and the platinum anode shown in Fig. 3. A time-averaged value of  $i_d$  was obtained by converting the output analog signals from an electric circuit including the spherical cathode into digital signals at a sampling rate of 500 s<sup>-1</sup> for 40–80 s and calculating an average value of these signals with a digital computer. Figure 4 shows a calibrated relation of  $i_d$  vs.  $U_p$  for the spherical cathode.

To measure the flow rate of circulating liquid in the column, the spherical cathode was installed on the radially moving device fitted to the column, and the voltage necessary to measure  $i_d$  was applied between the spherical cathode and an anode of platinum foil fixed on the inside wall of the column. The output signals from the electric circuit were sampled at a rate of 500 s<sup>-1</sup> for 80 s, and converted to the liquid velocities by the relation given in Fig. 4. These velocities were averaged to give  $U_{La}$ . In the annulus small bubbles of sizes below 4 mm were observed, and when these bubbles rapped the spherical cathode the values of the output signals dropped sharply. These signals were excluded from the data used to evaluate

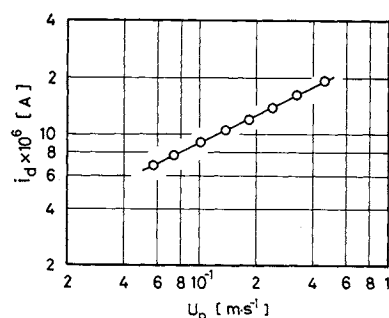


Fig. 4. Diffusion current vs. probe velocity.

$U_{La}$ .  $U_{La}$  obtained at six measuring points in radial direction of the annulus were multiplied by  $(1 - \epsilon_a)$  and used to evaluate  $Q_L$  by graphical integration.

### 2.3 Gas holdup and reversal losses

Axial distributions of static pressures in the draught tube and the annulus were measured: in the annulus the static pressure differences between pressure taps installed on the column wall were measured, and in the draught tube the static pressure difference between a reference pressure tap in the base plate and a 2 mm Prandtl tube inserted in the draught tube. Gas holdups and reversal losses were found from Eqs. (2)–(5) with the measured static pressure distributions,  $\tau_{wi}$  and  $\tau_{wa}$ , where  $\tau_{wi}$  and  $\tau_{wa}$  were calculated by using Takamatsu's results<sup>15)</sup> and the friction factor for single-phase liquid flow,<sup>5)</sup> respectively.

## 3. Results and Discussion

### 3.1 Static pressure distribution and reversal losses

Figure 5 shows  $\Delta P_t$ , the static pressure difference detected by a pressure transducer, two conduits of which were connected to the pressure tap of the base plate of the column and to an arbitrary pressure tap of the column or the Prandtl tube inserted into the draught tube. The static pressure  $P$  at a position whose height from the base plate is  $y$  is related to  $\Delta P_t$  by Eq. (12).

$$\Delta P_t = (P + \rho_L g y) - P_B \quad (12)$$

where  $P_B$  is the static pressure at the base plate of the column. An approximately linear relationship exists between  $\Delta P_t$  and  $y$  except for the lower part of the draught tube, where the entering fluid contracts and shows a lower value of  $\Delta P_t$  than that expected in the case without contraction. Therefore, to obtain the static pressure difference  $(P_4 - P_1)$  in Eq. (5) without the effect of contraction, the section of the straight-line plot of  $\Delta P_t$  vs.  $y$  is extrapolated to the lower end of the draught tube as shown in Fig. 5. The reversal loss at the lower end of the draught tube,  $\Delta P_b$ , was calculated from the measured value of  $(P_4 - P_1)$  and Eq. (5).

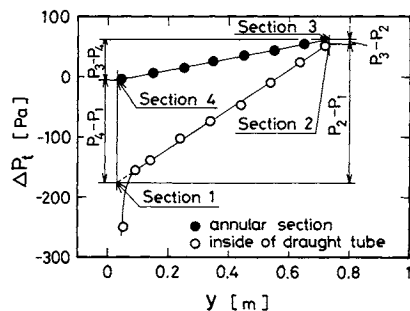


Fig. 5.  $\Delta P_i$  vs.  $y$ .

Figure 6 shows that  $\Delta P_i$  increases with increase in  $\bar{U}_{La}/(1-\epsilon_a)$ , and the column of  $D_i=0.082$  m gives relatively low values of  $\Delta P_i$  compared to those observed in columns having  $D_i$  other than 0.082 m.

Figure 7 shows that when the distance  $L$  between the lower end of the draught tube and the base plate of the column is increased from 2 cm to 4.2 cm ( $L/D_i=0.24-0.5$  in the column of  $D_o=0.14$  m and  $D_i=0.082$  m),  $\Delta P_i$  decreases drastically, but further increase in  $L$  has no effect on  $\Delta P_i$ . This dependence of  $\Delta P_i$  on  $L$  is similar to the results given by Blenke *et al.*<sup>3)</sup> for single-phase liquid flow. Figure 7 also shows that with the same column dimensions and the same liquid velocity, the more viscous liquid gives larger value of  $\Delta P_i$ , and the effect of  $D_o$  on  $\Delta P_i$  is small if  $D_i/D_o$  and  $L/D_i$  are kept constant.

The liquid flow under the draught tube is visualized by the paths of tiny bubbles entrained by the liquid flow from the annulus into the draught tube. This observation shows that the liquid flow from the annulus converges drastically under the lower end of the draught tube with change in its flow direction and then diverges into the draught tube. Therefore,  $\Delta P_i$  is assumed to be expressed in the form of loss due to the convergence-divergence flow model where the effect of flow reversal on  $\Delta P_i$  is included in an effective width  $h_i$  of the gas-liquid flow path under the lower end of the draught tube shown in Fig. 1.

$$\Delta P_i = 0.45 \left[ \frac{\rho_L}{2} \left\{ \frac{S_a \bar{U}_{La}}{\pi D_1 h_i (1-\epsilon_a)} \right\}^2 \right] \left( 1 - \frac{\pi D_1 h_i}{S_a} \right) + \frac{\rho_L}{2} \left( \frac{\bar{U}_{Li}}{1-\epsilon_i} \right)^2 \left\{ \frac{S_i (1-\epsilon_i)}{\pi D_1 h_i (1-\epsilon_a)} - 1 \right\}^2 \quad (13)$$

where  $(\pi D_1 h_i)$  is the minimum cross-sectional area of gas-liquid flow under the lower end of the draught tube.  $h_i$  was evaluated by Eq. (13) using measured values of  $\Delta P_i$ ,  $\bar{U}_{Li}$ ,  $\bar{U}_{La}$ ,  $\epsilon_i$  and  $\epsilon_a$ , and was correlated with column dimensions, flow conditions and liquid properties for columns of  $(L/D_i) \leq 0.5$ .

$$\frac{h_i}{D_o} = 0.187(1-\epsilon_a)^{-0.310} \left( \frac{\bar{U}_{Li}}{\sqrt{g D_i}} \right)^{0.310} \left( \frac{L}{D_i} \right)^{0.405}$$

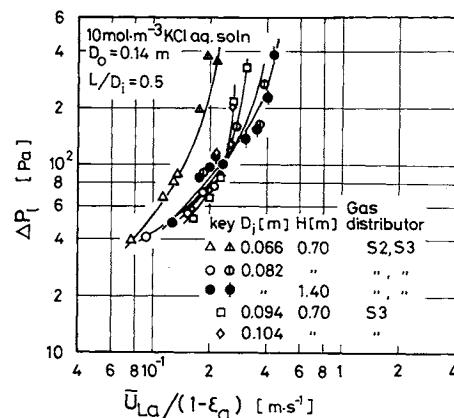


Fig. 6. Effect of  $\bar{U}_{La}/(1-\epsilon_a)$  and  $D_i$  on  $\Delta P_i$ .

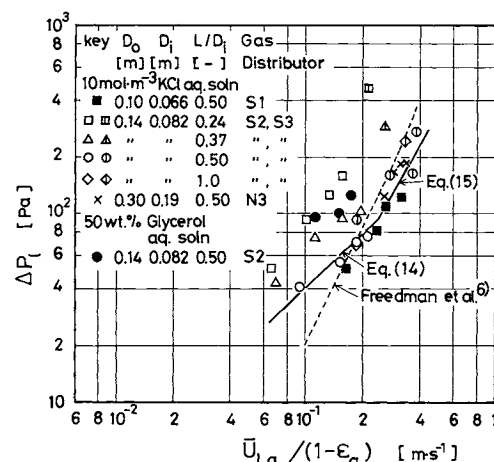


Fig. 7. Effect of  $\bar{U}_{La}/(1-\epsilon_a)$ ,  $\mu_L$ ,  $L$  and column diameter on  $\Delta P_i$  (Lines: estimated for air-10 mol·m<sup>-3</sup> KCl aq. soln system in column of  $D_o=0.14$  m,  $D_i=0.082$  m and  $L/D_i=0.50$ ).

$$\times \left( \frac{S_a}{S_o} \right)^{0.778} \left( \frac{S_i}{S_o} \right)^{0.176} \left( \frac{g \mu_L^4}{\rho_L \sigma_L^3} \right)^{-0.0218} \quad (14)$$

for  $0.710 \leq \phi \leq 1$

$$\frac{h_i}{D_o} = 1.05 \left( \frac{L}{D_i} \right)^{0.724} \left( \frac{S_a}{S_o} \right)^{2.28} \left( \frac{S_i}{S_o} \right)^{1.08} \left( \frac{g \mu_L^4}{\rho_L \sigma_L^3} \right)^{-0.0142} \quad (15)$$

for  $1 < \phi \leq 1.31$

where  $\phi$  is the ratio of the right side of Eq. (14) and that of Eq. (15),

$$\phi = 0.178(1-\epsilon_a)^{-0.310} \left( \frac{\bar{U}_{Li}}{\sqrt{g D_i}} \right)^{0.310} \left( \frac{L}{D_i} \right)^{-0.319} \times \left( \frac{S_a}{S_o} \right)^{-1.50} \left( \frac{S_i}{S_o} \right)^{-0.904} \left( \frac{g \mu_L^4}{\rho_L \sigma_L^3} \right)^{-0.0076} \quad (16)$$

The average errors of estimations of  $\Delta P_i$  with Eqs. (13) and (14) and with Eqs. (13) and (15), respectively, were 13.5% for 68 data and 34.9% for 40 data. For the estimation of  $\Delta P_i$  in columns of  $(L/D_i) > 0.5$ , it is recommended to estimate  $h_i$  with Eq. (14) or Eq. (15).

by assuming  $(L/D_i)=0.5$ .

Figure 7 shows that  $\Delta P_L$  values estimated with the equation of Freedman *et al.*<sup>6)</sup> agree relatively well with those observed in 10 mol·m<sup>-3</sup> KCl aq. soln and at  $L/D_i=0.5$ , but not with those observed in more viscous liquid or those observed in a column of  $(L/D_i)<0.5$ .

$\Delta P_u$  was calculated with Eq. (3) and the measured value of  $(P_3 - P_2)$ . Figure 8 shows that  $\Delta P_u$  is roughly equal to  $\rho_L \{ \bar{U}_{Li} / (1 - \varepsilon_i) \}^2 / 2$  in the column of  $(D_i/D_o) < 0.6$ . However, this relation does not hold if the value of  $D_i/D_o$  is increased further or if the clear liquid height is lowered to the level of the upper end of the draught tube, where  $\Delta P_u$  increases drastically.

The liquid flow near the upper end of the draught tube was visualized by paths of tiny pieces of paper dispersed in liquid and entrained by the liquid flow. This observation shows that the liquid flow from the draught tube diverges to the space above the draught tube with change in its flow direction as shown in Fig. 1 and then converges into the annulus. Therefore,  $\Delta P_u$  was assumed to be expressed in the form of loss of the divergence-convergence flow model where the effect of flow reversal on  $\Delta P_u$  was considered on deciding numerical constants in the model equation with experimental data of  $\Delta P_u$ .

$$\Delta P_u = 0.685 \rho_L \left( \frac{\bar{U}_{Li}}{1 - \varepsilon_i} \right)^2 \left\{ 1 - \frac{S_i(1 - \varepsilon_i)}{S_o(1 - \varepsilon_f)} \right\} + 0.945 \rho_L \left( \frac{\bar{U}_{La}}{1 - \varepsilon_a} \right)^2 \left\{ 1 - \frac{S_a(1 - \varepsilon_a)}{S_o(1 - \varepsilon_f)} \right\} \quad (17)$$

where  $\varepsilon_f$  is the gas holdup over the upper end of the draught tube,

$$\varepsilon_f = \{ D_1^2 \varepsilon_i + (D_o^2 - D_1^2) \varepsilon_a \} / D_o^2 \quad (18)$$

The average error of estimating  $\Delta P_u$  by Eq. (17) was 29.1% for 102 data.

### 3.2 Gas holdup in draught tube, $\varepsilon_i$

$\varepsilon_i$  was found from Eq. (2) with data of  $(P_1 - P_2)$  and  $\tau_{wi}$  estimated from Takamatsu's result.<sup>15)\*1</sup> Figure 9 shows that  $\varepsilon_i$  increases with increasing gas velocity  $\bar{U}_{Gi}$  in draught tube. As  $\bar{U}_{Li}$  affects  $\varepsilon_i$ , the effect of column dimensions on  $\varepsilon_i$  is not clear from Fig. 9. However, when a single nozzle or a single orifice plate is used as a gas distributor,  $\varepsilon_i$  is lower than that for a perforated plate. Therefore, different correlations of  $\varepsilon_i$  with experimental conditions were tried for the perforated plate and for other gas distributors.

From the theoretical equations of Nicklin<sup>13)</sup> for  $\varepsilon_i$  in finite liquid flow, Eq. (19) is derived.

$$(1/\varepsilon_i) - (1/\varepsilon_{i0}) = (\bar{U}_{Li} / \bar{U}_{Gi}) \quad (19)$$

where  $\varepsilon_{i0}$  is gas holdup with no liquid flow. Since  $\varepsilon_i$  obtained for single nozzle and single orifice plate did

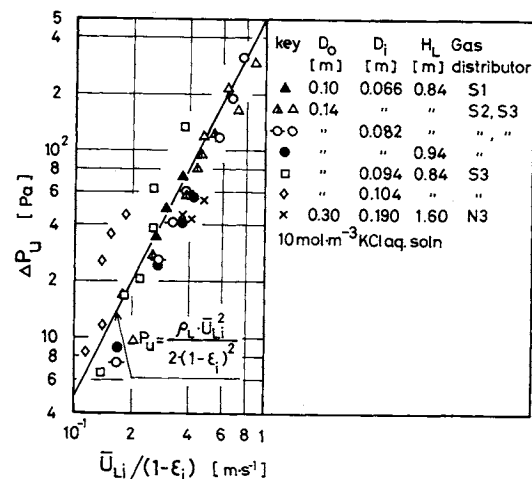


Fig. 8. Effect of  $\bar{U}_{Li}/(1 - \varepsilon_i)$  on  $\Delta P_u$ .

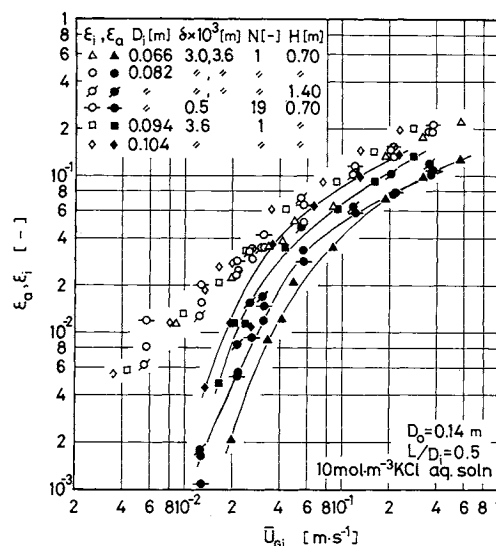


Fig. 9. Effect of  $\bar{U}_{Gi}$  on gas holdup.

not agree with Eq. (19),  $(1/\varepsilon_i - 1/\varepsilon_{i0})$  was correlated with  $\bar{U}_{Gi}$ ,  $\bar{U}_{Li}$ ,  $D_i$  and the liquid properties by Eq. (20) with average error of 8.2% for 108 data, where  $\varepsilon_{i0}$  was estimated by the equation of Akita *et al.*<sup>1)</sup>

$$\frac{1}{\varepsilon_i} - \frac{1}{\varepsilon_{i0}} = 0.00894 \left( \frac{\bar{U}_{Li}}{\bar{U}_{Gi}} \right)^{1.27} \left( \frac{\bar{U}_{Li}}{\sqrt{g D_i}} \right)^{-0.260} \times \left( \frac{D_i^2 g \rho_L}{\sigma_L} \right)^{0.229} \left( \frac{g \mu_L^4}{\rho_L \sigma_L^3} \right)^{-0.101} \quad (20)$$

$\varepsilon_i$  values observed for perforated plates were compared with those estimated by the method of Koide *et al.*<sup>9)</sup> and this showed that  $\varepsilon_i$  estimated agreed with values observed with average error of 10.3% for 18 data.

### 3.3 Gas holdup in annulus, $\varepsilon_a$

$\varepsilon_a$  was found from Eq. (4) with measured value of  $(P_3 - P_4)$  and  $\tau_{wa}$  was assumed to be given by Eq. (21) for single-phase liquid flow.<sup>5)</sup>

$$\tau_{wa} = 0.085 (D_e \bar{U}_{La} / \nu_L)^{-1/4} (\rho_L \bar{U}_{La}^2 / 2) \quad (21)$$

\*1 See Appendix.

Figure 9 shows that  $\varepsilon_a$  increases with increasing  $\bar{U}_{Gib}$  ( $D_i/D_o$ ) and liquid viscosity. When the froth height  $H_f^{*2}$  from the upper end of the draught tube is decreased, many bubbles are entrained by the liquid flow before they reach top of the froth, and so  $\varepsilon_a$  increases. Based on these observations, the following empirical equation of  $\varepsilon_a$  was proposed. It correlates  $\varepsilon_a$  with experimental conditions with an average error of 28.1% for 97 data.

$$\frac{\varepsilon_a}{(1-\varepsilon_a)^4} = 26.3 \varepsilon_i^{1.93} \left( \frac{\bar{U}_{La}}{\sqrt{gD_e}} \right)^{0.552} \left( \frac{S_o}{S_a} \right)^{1.25} \times \left( \frac{S_a}{\pi D_1 H_f} \right)^{0.614} \left( \frac{H_L}{H} \right)^{0.814} \left( \frac{g \mu_L^4}{\rho_L \sigma^3} \right)^{0.0328} \quad (22)$$

Estimated values of  $\varepsilon_a$  by Eq. (22) agree relatively well with the experimental values of  $\varepsilon_a$  observed by Bohner *et al.*<sup>4)</sup>

### 3.4 Flow rate of circulating liquid, $Q_L$

Figure 10 shows that  $Q_L$  in the 0.14 m dia. column increases with increasing  $\bar{U}_G$  up to  $\bar{U}_G \approx 0.02 \text{ m} \cdot \text{s}^{-1}$  but does not increase so much in the range of  $\bar{U}_G > 0.02 \text{ m} \cdot \text{s}^{-1}$ . When  $D_i/D_o$  is kept constant,  $Q_L$  increases with increasing column diameter for a given gas velocity and also with increasing draught tube length, as the driving force of liquid circulation increases. When the liquid viscosity is higher,  $Q_L$  becomes smaller due to increasing value of  $\Delta P_i$ . Figure 10 also shows that  $Q_L$  is larger when gas is dispersed by a perforated plate rather than a single orifice plate or a single nozzle. This might be due to the fact that the perforated plate generates smaller bubbles than the single nozzle or the single orifice plate and then  $\varepsilon_i$  becomes higher.

Figure 11 shows that the maximum value of  $Q_L$  is observed at  $D_i/D_o \approx 0.6$  for constant values of  $\bar{U}_G$  and  $D_o$ . Figure 12 shows that  $Q_L$  increases with increasing  $L$  in the range of  $L/D_i \leq 0.5$  but that further increase in  $L$  has almost no effect on  $Q_L$ . The reason is that  $\Delta P_i$  decreases with increasing  $L$  in the range of  $L/D_i \leq 0.5$  but further increase in  $L$  has no effect on  $\Delta P_i$ . The effect of the clear liquid height from the upper end of the draught tube on  $Q_L$  becomes small in the range of  $(H_L - L - H)/D_i > 0.7$ .

### 3.5 Estimation of $Q_L$ and comparison with previous works

$Q_L$  at a given gas velocity can be estimated by solving Eq. (6) or (9) numerically, as each term in Eqs. (6) and (9) is expressed as a function of  $Q_L$ ,  $\bar{U}_G$ , the column dimensions,  $H_L$  and the liquid properties.  $\Delta P_i$ ,  $\Delta P_w$ ,  $\varepsilon_i$ ,  $\varepsilon_a$ ,  $\tau_{wi}$  and  $\tau_{wa}$  are expressed, respectively, by Eqs. (13), (17), (20), (22), (A1) and (21). Though most experiments were carried out in columns with a

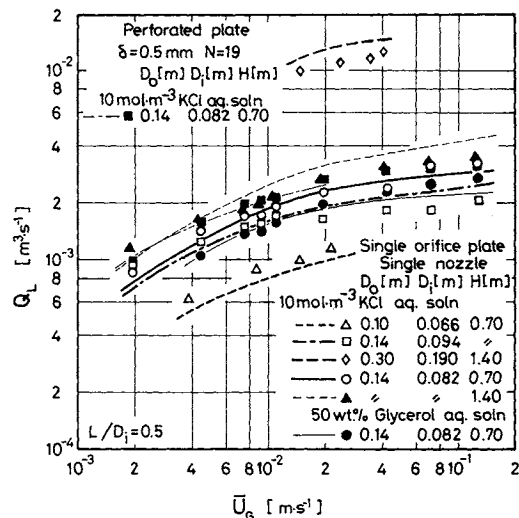


Fig. 10. Effect of gas velocity, column diameter, draught tube length and liquid viscosity on  $Q_L$  (lines: values of  $Q_L$  estimated by Eq. (6)).

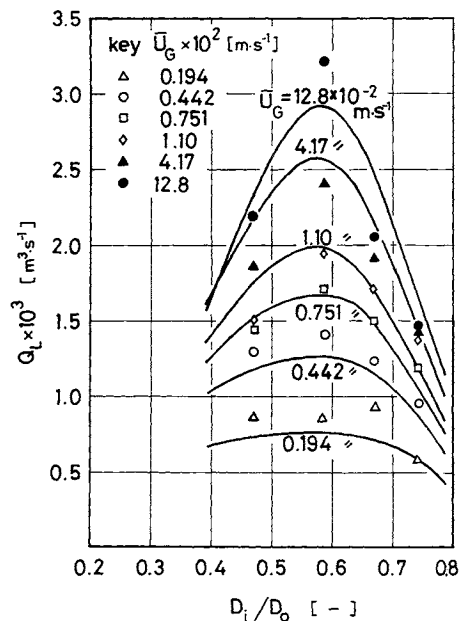


Fig. 11. Effect of  $D_i/D_o$  ratio on  $Q_L$  (Solid lines: Values of  $Q_L$  estimated by Eq. (6) for air-10 mol  $\text{m}^{-3}$  KCl aq. soln system in column of  $D_o = 0.14 \text{ m}$ ,  $H = 0.70 \text{ m}$  and  $L/D_i = 0.50 \text{ m}$ . Gas distributors: S2 for  $U_G \leq 0.11 \text{ m} \cdot \text{s}^{-1}$  and S3 for  $U_G > 0.11 \text{ m} \cdot \text{s}^{-1}$ ).

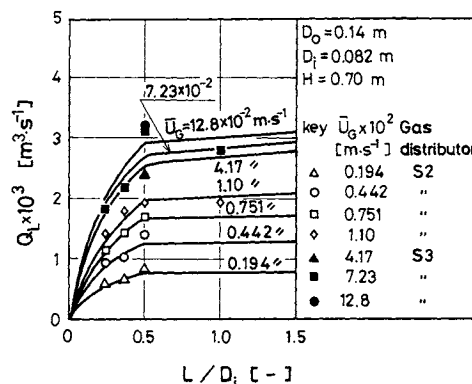


Fig. 12. Effect of  $L$  on  $Q_L$  (Solid lines: values of  $Q_L$  estimated by Eq. (6) for air-10 mol  $\text{m}^{-3}$  KCl aq. soln system).

\*2 Calculation of  $H_f$  is shown in Appendix.

single orifice plate or a single nozzle as a gas distributor, the above method of estimating  $Q_L$  can be applied to the columns with a perforated plate if  $\varepsilon_i$  is estimated by the method of Koide *et al.*<sup>9)</sup> instead of Eq. (20). Estimated values agree relatively well with those observed, as shown in Figs. 10–12, and the average error of estimating  $Q_L$  was 13% for 127 data.

The average velocity of circulating liquid  $\bar{U}_{LC}$  defined by Eq. (1) was calculated from  $Q_L$  observed in the previous works<sup>4,6,8)</sup> and in this work as shown in Fig. 13.  $\bar{U}_{LC}$  observed by Bohner *et al.*<sup>4)</sup> ( $D_i/D_o=0.572$  and  $L/D_i=0.439$ ) and Freedman *et al.*<sup>6)</sup> ( $D_i/D_o=0.589$  and  $L/D_i=0.444$ ) roughly agreed with those observed in this work ( $D_i/D_o \approx 0.6$  and  $L/D_i=0.5$ ).  $\bar{U}_{LC}$  values observed at different column diameters in this work agree well with each other if  $D_i/D_o$  and  $L/D_i$  are kept nearly constant. Figure 13 also shows that the values of  $\bar{U}_{LC}$  estimated by the equation of Miyauchi *et al.*<sup>12)</sup> for a bubble column without draught tube are much smaller than those in a column with draught tube. A rough estimation of the circulation time  $t_c (=V_L/Q_L \approx 2H_L/\bar{U}_{LC})$  can be made by using Fig. 13.

Figure 14 shows that  $(\Delta P_i + \Delta P_u)$  and  $(\Delta P_{fa} + \Delta P_{fi})$  are about 70% and 30% of the total pressure drop, respectively, and  $\Delta P_i$  is about 35–65% of the total pressure drop. Therefore, disregarding  $\Delta P_{fa}$ ,  $\Delta P_{fi}$  and  $\Delta P_u$  on estimating  $Q_L$  by Eq. (6), which was done by Freedman *et al.*<sup>6)</sup> results in an overestimate of  $Q_L$  value as Gopal *et al.*<sup>8)</sup> have pointed out.

## Conclusions

1) The flow rate of circulating liquid  $Q_L$  in a bubble column with draught tube increases with increasing gas velocity, column diameter, draught tube length and  $L/D_i$  ratio below 0.5 and decreases with increasing liquid viscosity. However, the maximum value of  $Q_L$  is observed at  $D_i/D_o \approx 0.6$  for constant values of gas velocity and column diameter.

2) Empirical equations of gas holdups in draught tube and annulus and the pressure drops due to flow reversals, necessary for estimating  $Q_L$ , are proposed.

## Appendix

### A.1 Evaluation of $\tau_{wi}$

Koide *et al.*<sup>10)</sup> derived Eq. (A1),

$$\tau_w = \tau_{w0} (k_L/k_{L0})^3 \quad (\text{A1})$$

where  $k_L$  and  $\tau_w$  are, respectively, the local mass transfer coefficient and the shear stress at the wall in gas-liquid two-phase flow. Takamatsu<sup>15)</sup> proposed empirical equations for  $(k_L/k_{L0})$  based on data found in a column of  $D_i=0.05$  m and air–20 mol·m<sup>-3</sup> KCl aq. soln system and for the range of  $\bar{U}_{Li}=0.254$ – $0.511$  m·s<sup>-1</sup>: for perforated plate ( $\delta=1$  mm,  $N=19$ ),

$$(k_L/k_{L0}) = 1.08 (\bar{U}_s/\bar{U}_{Li})^{1/2}, \quad 0 < \varepsilon_i \leq 0.05 \quad (\text{A2})$$

$$(k_L/k_{L0}) = 1.49 e^{1/8} (\bar{U}_s/\bar{U}_{Li})^{1/2}, \quad 0.05 < \varepsilon_i \leq 0.15 \quad (\text{A3})$$

and for single orifice plate ( $\delta=3$  mm,  $N=1$ ),

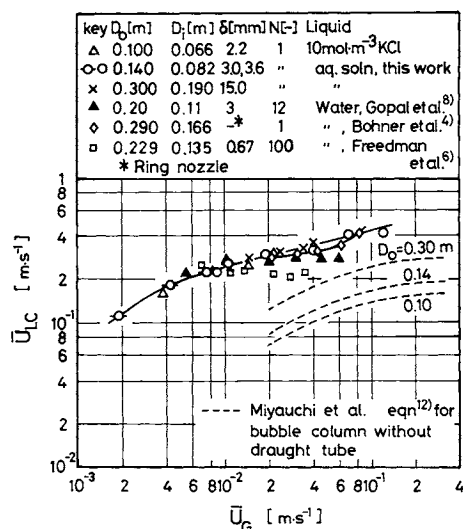


Fig. 13. Comparison of average velocities of circulating liquid observed in this work with those in previous works.

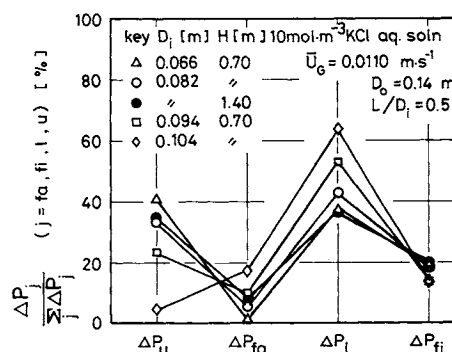


Fig. 14. Comparison of pressure drops due to friction and flow reversal.

$$(k_L/k_{L0}) = 1.57 e^{1/11} (\bar{U}_s/\bar{U}_{Li})^{1/2}, \quad 0 < \varepsilon_i \leq 0.15 \quad (\text{A4})$$

$\tau_{wi}$  was evaluated with Eqs. (A1)–(A4), where  $\tau_{w0}$  was calculated by using the Blasius friction factor.

### A.2 Calculation of $H_f$

The liquid volume  $V_L$  and the gas volume  $V_G$  in the column are given by Eqs. (A5) and (A6), respectively.

$$V_L = S_o H_L - V_d \quad (\text{A5})$$

$$V_G = S_o H_f \varepsilon_f + S_a H \varepsilon_a + S_i \varepsilon_i (H + L - H_N) \quad (\text{A6})$$

Average gas holdup  $\bar{\varepsilon}$  is given by Eq. (A7).

$$\bar{\varepsilon} = V_G / (V_G + V_L) \quad (\text{A7})$$

The height  $H_f$  of the aerated liquid from the base plate is given by Eq. (A8).

$$H_f = (V_G + V_L + V_d) / S_o \quad (\text{A8})$$

Referring to Fig. 1,  $H_f$  is given by Eq. (A9).

$$H_f = H_f - H - L \quad (\text{A9})$$

Substituting Eqs. (A5), (A6) and (A8) into Eq. (A9) and rearranging it gives the following equation.

$$H_f = \frac{H_L - H - L}{1 - \varepsilon_f} + \frac{S_a H \varepsilon_a + S_i \varepsilon_i (H + L - H_N)}{S_o (1 - \varepsilon_f)} \quad (\text{A10})$$

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## Nomenclature

$D$	= inner diameter	[m]
$D_e$	= $(D_o - D_i - 2t_w)$	[m]
$D_i$	= $(D_i + t_w)$	[m]
$g$	= gravitational acceleration	$[m \cdot s^{-2}]$
$H$	= length of draught tube	[m]
$H_F, H_f$	= level of aerated liquid, respectively, measured from base plate of column and from upper end of draught tube	[m]
$H_L$	= clear liquid height	[m]
$H_N$	= height of nozzle from base plate of column	[m]
$h_i$	= effective flow width of gas-liquid flow path shown in Fig. 1	[m]
$i_d$	= diffusion current	[A]
$k_L$	= local mass transfer coefficient at wall	$[m \cdot s^{-1}]$
$L$	= distance between lower end of draught tube and base plate of column	[m]
$N$	= number of nozzles or pores	[—]
$P_j$	= static pressure at section $j$ ( $j=1-4$ )	[Pa]
$\Delta P_b, \Delta P_u$	= pressure drop due to flow reversal at lower end and at upper end of draught tube, respectively	[Pa]
$\Delta P_f$	= frictional pressure drop	[Pa]
$\Delta P_t$	= static pressure difference detected by pressure transducer	[Pa]
$Q_G$	= gas flow rate	$[m^3 \cdot s^{-1}]$
$Q_L$	= flow rate of circulating liquid	$[m^3 \cdot s^{-1}]$
$S$	= cross-sectional area	$[m^2]$
$t_w$	= wall thickness of draught tube	[m]
$\bar{U}_G, \bar{U}_{Gi}$	= gas velocity based on cross section of column and draught tube, respectively, and based on average static pressure in column	$[m \cdot s^{-1}]$
$\bar{U}_L$	= average liquid velocity based on cross section	$[m \cdot s^{-1}]$
$U_{La}$	= liquid velocity in annulus	$[m \cdot s^{-1}]$
$\bar{U}_{LC}$	= average velocity of circulating liquid defined by Eq. (1)	$[m \cdot s^{-1}]$
$U_p$	= probe velocity	$[m \cdot s^{-1}]$
$\bar{U}_s$	= $\bar{U}_{Gi}/\epsilon_i - \bar{U}_{Li}/(1 - \epsilon_i)$	$[m \cdot s^{-1}]$
$V_d$	= $H\pi\{D_i + 2t_w\}^2 - D_i^2/4$ , volume of draught tube	$[m^3]$
$V_G, V_L$	= volumes of gas and liquid, respectively	$[m^3]$

$\delta$	= pore or nozzle diameter	[m]
$\epsilon$	= gas holdup	[—]
$\epsilon_f$	= gas holdup defined by Eq. (18)	[—]
$\epsilon_{i0}$	= gas holdup in draught tube at $\bar{U}_{Li}=0$	[—]
$\mu_L$	= liquid viscosity	$[Pa \cdot s]$
$\nu_L$	= liquid kinematic viscosity	$[m^2 \cdot s^{-1}]$
$\sigma_L$	= liquid surface tension	$[N \cdot m^{-1}]$
$\tau_w$	= shear stress at wall	[Pa]

## <Subscripts>

$a$	= annulus
est.	= estimated value
$i$	= draught tube
$o$	= column
obs.	= observed value
0	= single-phase liquid flow

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